



## INCREASED STIFFNESS OF VEHICLE STRUCTURE IN ACCIDENT

### CROSS REFERENCE TO RELATED APPLICATIONS

- 5 This is a continuation-in-part application of co-pending international application number PCT/DE 96/02120 (WO 97/18984, European Patent Doc. EP 0869878 B1) filed Nov. 7, 1996.

### BACKGROUND OF THE INVENTION

10 1. Field of the Invention:

The present invention relates generally to vehicle doors and, more particularly, to interengaging assemblies which structurally integrate all vehicle doors, when closed, with the vehicle roof, both side rails (sill portions) arranged along the vehicle floor, all pillars (post sections or pillar portions) and the flanges of door apertures of a vehicle body thereby distributing energy to all those vehicular members, lowering stress thereof, preventing passenger ejection and enhancing survival chance in the event of any collision (front, side and/or rear collision) or rollover.

2. Discussion of the Prior Art:

- 20 In order to formulate in single terminology a generalized definition for the proper term is presented:

**Definition:**

"series-connected doors"

"girder"

"window-guide channels" of window pane (glass)

"door cavity"

"door detachment"

"mating parts of interengaging assembly"

"engaging hole"

"vehicular couple"

**Proper Term:**

doors of one vehicle side are series-connected

panel, shell, beam etc. according to FEM and Technical Mechanics

window-pane tracks 6, 6B, 6.1, 6.2, 6.1B, 6.2B, 6.1a, 6.2a, 6.1aB, 6.2aB

space between the outer and inner panel of the door

vehicle door becomes detached from the vehicle body

mating parts of an interengaging assembly such as key & receptacle, hook & recess, hole & key or hook & rod

aperture, slot, oblong hole

two mating vehicular members, such as vehicle door & vehicle roof, vehicle door & side rail, vehicle door & flange (transition region) of vehicle body, vehicle door & pillar/s, vehicle door & vehicle door in engagement in the event of any collision and/or rollover

- 25 It is known in the prior art to provide interengaging assemblies to engage and/or clamp the vehicle door with the mating vehicular members, when the vehicle door is in closed position, thus distributing energy, lowering stress whilst enhancing survival chance only in the event of either mid-front collision or side collision of type „U2”, one of four types shown in Fig. 13.

However, all these conventional configurations do not take into account the failure of passenger protection due to the following problem cases in conjunction with disengagement of the mating parts of interengaging assemblies from each other in the event of all types of real-world collision or real-world rollover:

- 5     **A** Load cases **I** to **V** according to Technical Mechanics/FEM in real-world front, side and rear collisions;
- B** Wrong assumption of the prior art for the purpose of idealizing a general side energy „S” or „S<sub>1</sub>” to a single energy „S<sub>x</sub>” or „S<sub>x1</sub>”;
- C** Analogy between the state of non-contact and disengagement;
- 10    **D** Constant, small contour-clearance and assembly tolerance zones;
- E** Large clearances of interengaging assemblies;
- E1** The first inventions of interengaging assemblies, huge production costs and fatal injury in a real-world collision due to large clearances;
- E2** Large deformation of vehicle structure or door **8. 8B** in a real-world collision;
- 15    **E3** Large deformation of side rail **18** in a real-world collision;
- E4** Large deformation of upper member **8.17** of door frame and vehicle roof **17** in a real-world collision;
- E5** Intrusion of vehicle roof **17** in vehicle body **20** in a real-world rollover-accident; and
- E6** Clamping assemblies or adjustable interengaging assemblies to resolve problem case
- 20    **E**.

Problem case **A**: In order to idealize an impact force „2F<sub>1</sub>”, shown in Fig. 10A, imposed on a vehicle structure the following assumptions must be specified:

- let the vehicle structure be idealized by two symmetric vehicle halves subjected to an front impact force „2F” along the centre line.
- 25    Load case **I** in z-y plane in Fig. 5: The moment „M<sub>x</sub> = H\*h” about the x-axis is replaced by a pair of forces „H<sub>A</sub> = (H\*h)/l” with the lever arm of „l”. Employing the equilibrium condition for moments two forces of reaction are obtained: „V<sub>A</sub> = (V\*l<sub>C</sub>)/l” and „V<sub>B</sub> = -V<sub>A</sub> + V”. Acting in z-direction with respect to the sign are three shear forces: „-V”, „(H<sub>A</sub> + V<sub>A</sub>)” and „-(H<sub>A</sub> + V<sub>B</sub>)”. Under load of these forces the vehicle side, comprising all pillars,
- 30    series-connected doors **8, 8B** reinforced by impact elements and interengaging assemblies of those doors and pillars, is subjected to the bending moment along the y-axis.
- Load case **II** in z-x plane in Fig. 6: The force „V exerts bending moment „M<sub>zx</sub>” along the x-axis and rotating moment „M<sub>y</sub> = V\*b” about the y-axis acts as torsional moment along the vehicle side.
- 35    Load case **III** in x-y plane in Fig. 7: The A-pillar is under load of rotating moment „M<sub>xy</sub> = -H\*b”. The vehicle side is subjected to bending moment „M<sub>xy</sub>” along the y-axis and buckling force „H”.
- Subjected to the total stress of bending moments „M<sub>zx</sub>”, „M<sub>xy</sub>”, „M<sub>zy</sub>”, buckling force „H” and torsional moments „M<sub>z</sub>”, „M<sub>y</sub>” in the load cases **I** to **III**, the vehicle side, shown in Fig.
- 40    **8**, is deformed in a real-world front collision.
- By reversibly arranging the series-connected doors **8, 8B** the same load cases are obtained for real-world rear collision.

- Load case **IV** in x-y plane in Fig. 9: Under load of side impact energy „S” at impact angle „α = 27°” according to FMVSS 214 or in the event of real-world side collision the vehicle side is subjected to bending moment „M<sub>xyS</sub>” along the y-axis and lateral force „S<sub>y</sub>”.
- Load case **V** in z-x plane in Fig. 10: Under load of side impact energy „S” at impact angle „γ” or in the real-world side collision against a tree or highway column **22**, shown in Fig. 10A, 13, the vehicle side is subjected to bending moment „M<sub>zS</sub>” along the z-axis and lateral
- 50    force „S<sub>z</sub>”.

The total stress consists of the stresses in load cases IV and V.

Problem case B: The majority of the prior art is governed by the following assumptions:

- let clearances between mating parts of an interengaging assembly be neglected and
- let the load cases IV and V be idealized to a lateral energy „S<sub>x</sub>”, shown in Fig. 9, or „S<sub>x1</sub>”, shown in Fig. 10A, imposing on the *centre* of vehicle door, illustrated as collision type „U1”, shown in Fig. 13, despite four collision types „U1 to U4” and the collision type „U2” having the highest percentage of severe and fatal injuries. Nevertheless, car manufacturers and suppliers world-wide have adopted this idealized „S<sub>x</sub>” or „S<sub>x1</sub>” in inventions e.g. U.S. Pat. No. 4,307,911, U.S. Pat. No. 5,806,917, U.S. Pat. No. 5,518,290, whose shortcomings are mentioned in the following problem case E2.

Problem case C: Ref. to Figs. 11, 12 both end coils of compression-coil spring 19 are guided by two spring seats 19.1. Their utmost outer nodes „KN<sub>1</sub>” and „KN<sub>End</sub>” (not drawn) rest against both stops 19.3, where „i” represents the number of coils. To survey the rolling behaviour of end coil 19 on the lower spring seat 19.1 the end coil is idealized in elements by supporting springs in reference to the nodes and by the threshold value of the distance in the "state of rolling" „s < 0.1” mm. Fig. 12 illustrates the rolling behaviour in regard to the FEM data and test results marked with M in dependence on „F<sub>z</sub> = -790, -1000 and -3000 N”:

- According to test results „KN<sub>2</sub>” to „KN<sub>5</sub>” roll on the spring seat at „F<sub>z</sub> = -790 N”, but in the state of non-contact at „F<sub>z</sub> = -1000 and -3000 N”.
- According to FEM data the nodes in the following states are in dependence on „F<sub>z</sub>”:

F <sub>z</sub>	State of contact	State of rolling
-100	KN <sub>1</sub> , KN <sub>15</sub> , KN <sub>17</sub>	KN <sub>1</sub> to KN <sub>3</sub> , KN <sub>10</sub> to KN <sub>18</sub>
-250	KN <sub>1</sub> , KN <sub>19</sub> , KN <sub>20</sub>	KN <sub>1</sub> , KN <sub>15</sub> to KN <sub>23</sub>
-1415	KN <sub>1</sub> , KN <sub>17</sub> , KN <sub>19</sub> , KN <sub>20</sub> , KN <sub>30</sub> , KN <sub>31</sub> , KN <sub>33</sub> , KN <sub>34</sub>	KN <sub>1</sub> , KN <sub>15</sub> to KN <sub>35</sub>

The state of contact (engagement) of mating parts of interengaging assemblies, idealized by nodes of the rolling end coils and mating elements of the spring, can be transformed into the state of disengagement, when the force increases.

Problem case D: Recently in automotive industry, great efforts have been made to achieve (finish) a constant (uniform), small contour clearance between the outer door-contour "abcde" of vehicle door 8, 8B and the door aperture of vehicle body 20, shown in Fig. 5, in order to minimize flow noise and, particularly, to achieve sales success in co-operation with an overall impression of attractive design. In the state of assembly the contour clearance e.g. of AUDI ® vehicles is only 2.5 mm and of VW Passat ® 3.5 mm.

In order to meet the above-mentioned goal and to avoid rework or reject rate large assembly tolerances between the outer door-contour and the door aperture (opening) of vehicle body 20 must be designed.

Problem case E: The door lock 248, rigidly attached to vehicle door 8, and the striker 298, rigidly attached to pillar illustrated as B-pillar in Fig. 10A of U.S. Pat. No 4,307,911 representing the prior art, is provided with locking clearances in x-, y- and z-direction, thus ensuring the state of door locking and the normal operation of vehicle door. For the purpose of preserving the constant, small contour-clearance,

- the position „D<sub>a</sub>” to „D<sub>c</sub>” of each key 128a to 128c, rigidly attached to vehicle door 8, and the position „S<sub>a</sub>” to „S<sub>c</sub>” of mating receptacle 158a to 158c, rigidly attached to lower stiff panel 156 of side rail 18;

– the position „D<sub>n</sub>” of key 148, rigidly attached to vehicle door 8, and the position „B<sub>n</sub>” of mating receptacle 198, rigidly attached to pillar, must be provided with position-tolerances, larger than locking and assembly tolerances, in x-, y- and z-direction in order to avoid

- 5 1. interference with the locking operation of door lock 248 to striker 298 when closing vehicle door 8;
2. expensive reworking at the assembly line;
3. customer complaints due to disturbing noises associated with the small distances of overlaying coils, representing the mating parts of interengaging assemblies, denoted as  
10 „w ≤ 0.2 mm”, shown in Fig. 11; and
4. high reject rate due to different references of coordinate system of vehicle door, finished by two to three suppliers and transported to assembly line, and of vehicle body 20, finished at the assembly line. Huge costs are necessary to computerize design data of vehicle door and structure in data files, which must be evaluated by innovative programs  
15 to minimize those position-tolerances and reject rate, however, under the condition of the constant, small contour-clearance.

Problem case E1: According to the prior art the taper-formed key 148 and the mating receptacle 198 should be in engagement or form-locking connection to ensure energy-transmission from one pillar to the other.

20 Because receptacle 198 and striker 298 are formed together in one piece, an adjustment of receptacle 198 changes the position of striker 298 to the door lock 248 as well as the clearance therebetween, which becomes too large or small. In order to properly latch and lock the vehicle door to vehicle structure the "interengaging" assembly is provided with large tolerance zones, thus violating the condition of the aforementioned feature.

25 When a vehicle is laterally crashed by a truck, the key 148 can disengage from mating receptacle 198 due to large clearance so the remaining energy totally deforms the vehicle door, whose intrusion can fatally injure the driver.

According to the prior art shown in Fig. 1A, contour tongues 16.1 should be in engagement with contour grooves 16.2 in order to integrate vehicle door 8, 8B into side rail  
30 18, vehicle roof 17 and B-pillar in a side collision. Without "interengaging" assembly of the vehicle door and B-pillar, the normal operation of vehicle door would be possible if the outer door-contour "abcde" were square. Regarding the recent contour design, shown in Figs. 5 and 18, the line "ab" is generally curve-shaped, line "bc" of front door upwardly inclined ( $\beta > 90^\circ$ ) or generally curve-shaped and line "bc" of rear door generally S-shaped,  
35 so contour grooves 16.2 would interfere with contour tongues 16.1 when closing the vehicle door. Furthermore, to sustain large impact energy it is necessary to reinforce the wide contour groove by an element which, unfortunately, can't be attached to the narrow upper member 8.17 of door frame.

According to the U.S. Pat. No. 3,819,228 a bulky "engaging" bolt rigidly attached to a  
40 stiff inner panel of vehicle door 8 projects through a hole of a stiff element attached to side rail 18 when the door is in closed position. The problem of large tolerance zones remains unresolved. Moreover, the overall stylish impression spoilt by a bulky "engaging" bolt will, doubtless, not be beneficial to sales. When stepping in or out of the vehicle body while cleaning or repairing, the person can injure himself when stumbling over this bulky bolt.  
45 When closing the door the danger of damage to clothing and injury to passengers, particularly when it is dark, is apparent.

Problem case E2: Under the load of force „F<sub>1</sub>”, shown in Fig. 10A, in an approx. 30° inclined, offset front collision against another car the vehicle structure, totally deformed, is deflected, in great extent, in the opposite x-direction and in the y-direction thus resulting in

disengagement of the catching hook 148, rigidly attached to the impact beam 1, 1B of driver-door, and the door lock 248 from the mating recess 198 and striker 298, all of which are rigidly attached to the B-pillar, respectively, in association with the reduction of the distance between the A- and B- pillar from 860 mm to 490 mm in the y-direction and the collapse of passenger protection. Later on, the remaining energy totally deforms the driver-door too. If the car rolls over, the driver would be ejected therefrom.

In a real-world side collision of another car into a tree, great energy totally deformed the vehicle side whose intrusion fatally injured both passengers. Obviously, the lateral force, deviating from the idealized force „ $S_{X1}$ ”, can never force catching hook 148 to penetrate into recess 198 in order to define an "interengaging" assembly. Both and other real-world accidents resulting in the collapse of vehicle structure linked to severe/fatal injuries verify the shortcomings of any patent valid only for survival chance under load of an idealized force „ $S_{X1}$ ”, shown in Fig. 10A.

As exemplified by U.S. Pat. No. 5,518,290, the tapered (wedge shaped) keys, located on the front, rear, upper and lower edges of the door, engage mating receptacles, located on the door jamb, the roof and side rail, when the door is closed. Four clearances, associated with four faces thereof, and a depth clearance are accounted for loose engagement of each tapered key with the mating receptacle. On the use of six interengaging assemblies the structural connection of door 8 with vehicle body 20 has  $6 \times 5 = 30$  clearances plus three clearances between the stud 298 and catch of door lock 248, shown in Fig. 10A, thus totalling 33 clearances, which must be designed large in order to avoid the interference of interengaging assemblies in each other. The drawback of large clearances should be resolved by injecting hardenable resin in the receptacles. Huge costs result from stopping the assembly line, meticulously measuring all 33 clearances, which must be adjusted to minimum tolerances, and repairing when the remaining resin smears the vehicle body and/or resin spills thereover. Disadvantageously, hardenable resin coatings are incapable of sustaining great forces in an accident.

When the vehicle body deflects in the opposite x-direction, the interengaging assemblies with large clearances are disengaged. In general, large clearances are responsible for the collapse of vehicle structure in accidents.

As exemplified by U.S. Pat. No. 4,676,524, a pair of vertically supporting window-guide channels, rigidly mounted in both vehicle doors 8 of a convertible car is in abutting, "engaging" relationship with both termini of upper member of cowl, when both vehicle doors are in closed position, owing to a pair of "interengaging" assemblies, each of which consists of

1. a receptacle of the terminus of the upper member and a locking mating tip of key of the window-guide channel pressing therein in the first embodiment; or
2. a king-size hole of the terminus of the upper member and a mating key of the window-guide channel having a mushroom-shaped head being in free connection therewith in the second embodiment

for the purpose of enhancing survival chance on rollover.

When the convertible car rolls over,

1. great shear force fractures each locking tip of the key; or
2. great impact energy totally deforms each "interengaging" assembly, whose key and king-size hole are in disengagement,

thereby totally deforming the cowl and pair of window-guide channels.

The stiffness of an open roof of a convertible car, merely supported by a pair of pillars in force-locking or free connection with one pair of small-size window-guide channels, is

– very low, thereby resulting in fatality on a real-world rollover thereof;

– lower than that of a rotatable, stiff rollover bar;

- far lower than that of the closed roof 17 supported by two pairs of pillars and
- substantially far lower than that of the closed roof 17 strongly supported by three pairs of reinforced pillars.

Problem case E3: Due to great energy in a real-world side collision against column 22 of a central barrier, shown in Figs. 10A, 13, on a highway

- large deformation of side rail 18 and rear section of a vehicle, opposite to x-direction, caused the disengagement of the driver's less deformed vehicle door 8 from vehicle structure and later on
- the vehicle rolled over three times across the highway and down-hill, thus totally deforming vehicle structure, doors 8, tailgate-door 8T, out of which both rear passengers were hurled, and, alternately, opening and closing both vehicle doors 8, out of which both front passengers were hurled out.

Grass 70 clamped between each pillar and each vehicle door 8, shown in Fig. 8, was an evidence for the alternate opening and closing of both vehicle doors 8 during the rollovers.

In a side collision of a car into a tree great energy totally deformed vehicle door 8, whose intrusion severely/fatally injured the passengers.

In a collision of another car into a hill great energy totally deformed the right side rail 18 thus resulting in the disengagement of the door lock 248 and, if provided, interengaging assemblies too and later on totally deforming vehicle structure during rollover. The driver was hurled out of this car.

Problem case E4: In a front collision or crash test impact energy deforms, in general, upper member/s 8.17 of door frame/s outwards and vehicle roof 17 upwards, thereby creating a gap „o”, shown in Fig. 8. and preventing front vehicle door/s 8, 8B and/or vehicle roof 17 from transmitting energy to vehicle body 20.

Three different states of deformation are reproduced in three crash tests, conducted by ADAC, of the German vehicles of the same type 40 % offset crashed at the same speed of 50 km/h against

- a very stiff barrier,
- a deformable barrier and
- another vehicle of the same type

because the uniform load, deformable property of two colliding masses, impact condition etc. are different. The gap „o” in three different sizes, shown in Fig. 8, verifies the above-mentioned thesis of non-transmission of energy.

In a side collision impact energy deforms, in general, upper member/s 8.17 of door frame/s inwards thereby inflicting injuries on head.

Problem case E5: During the rollover of a car, impact energy totally deformed vehicle roof 17 whose intrusion severely or fatally injured both front passengers, whose heads were, definitely, crushed by falsely deployed airbags, and the remaining energy totally deformed vehicle body 20 and doors 8, 8B, 8T, 8x.

Problem case E6: Responsive to problem case E, a clamping assembly illustrated in Fig. 1B comprises

- a stiff hook of stiff ledge 25.2 rigidly mounted to lower door frame 8.18 and
- a thin mating panel of a stiff plate 25.1, rigidly attached along sill rail 18, serving as a site of predetermined fracture.

In excess of predetermined value in a real-world side accident, the mating parts 25.1, 25.2 of interengaging assemblies are in the state of clamping to ensure the permanent engagement of lower door frame 8.18 with sill rail 18 in order to resolve the problem of passenger ejection. Load cases I to III, V and problem cases E2 to E5 remain unresolved.

Furthermore, there is no space to house both mating parts 25.1, 25.2 in vehicle roof 17 and upper member 8.17 of door frame subjected to lateral load „F<sub>o</sub>” in real-world accidents. The lack of interengaging assemblies became obvious on the rollover of a sport car, which plunged seven meter downwards and crashed with vehicle roof 17 at a lower level of an underpass in Wiesbaden City thus totally deforming vehicle roof 17, body 20 and both upper members 8.17 of door frames during rollover, where the remaining energy was transmitted through both head rests, integrated into the respective seat backrests, to the vehicle floor, thereby reducing the AIS of both passengers. AIS is an international acronym of Abbreviated Injury Severity ranging from 0 (no injury) to 6 (fatality).

Responsive to problem case E, adjustable and/or door locks are provided for interengaging assemblies, whose adjustable and/or latchable keys are bolted to the B- or C-pillar, facing the termini of both reinforcing beams 1, 7 or 1B, 7B, and whose mating receptacles are arranged thereto. Both plates 5.1, 5.2 of each hinge of vehicle door are provided with a rivet, serving as key, and an oblong mating hole. Owing to this feature load cases I to IV are resolved, but load case V and problem cases E3 to E5 remain unresolved.

Evidently, due to load cases I to V and all problem cases B, E, E1 to E5 "interengaging" assemblies of the remaining prior art are unsuitable for the purpose of energy-transmission and distribution by means of the integration of vehicle doors 8, 8B, 8T into the vehicle body 20, in conjunction with five tolerance zones proposed by U.S. Pat. No. 5,297,841, U.S. Pat. No. 4,307,911 and eight tolerance zones proposed by U.S. Pat. No. 5,806,917.

None of the above-mentioned configurations offer the simplicity of the present invention in manufacturing the interengaging assemblies and ensuring the engagement therefor associated with the integration of the vehicle doors with the vehicle body in order to increase the stiffness of the vehicle structure and to prevent passengers from being hurled out of the vehicle, particularly, in the event of rollover.

## SUMMARY OF THE INVENTION

Accordingly, the principle object of the present invention is to overcome the deficiencies of the prior art by providing engagement for interengaging assembly having large clearances, which are necessary in car manufacturing and door assembly, in order

- to protect passengers against ejection from the vehicle body and/or intrusion of vehicular member and
- to increase the vehicular stiffness

in the event of any collision and/or rollover. These interengaging assembly are arranged to the corresponding vehicular couples (vehicular member & mating vehicular member).

This principle and other objects of the present invention are accomplished by the following features (proposals):

- minimum tolerances by installing and adjusting the engaging keys from outside to tightly mate the receptacles thereby ensuring the connection of the doors with all vehicular members of vehicle body 20 such as pillars, vehicle roof 17, flange 21, a pair of side rails 18, fastened to vehicle floor, in any collision and/or on rollover;
- interengaging assemblies with adjusting mechanisms such as holes & keys 15.1 to 15.5a, 15.7, 15.8, hooks 15.6 & reinforcing rod 17.1d and holes & keys 30 to 37, shown in Figs. 1, 3, 3A, 4, 4A and 14 to 18;
- window-guide channels to accommodate the engaging parts;
- space-saving, inexpensive design for engaging parts;

- arrangement of interengaging assemblies of a vehicular couple in at least two operating planes thus making the strict restriction of minimum tolerances less significant;
- arrangement of an U-shaped extension member having keys in the common pillar of the series-connected vehicle doors, whose holes mate with the keys to ensure the engagement owing to constrained deformation thereof.

Despite the failure of the prior art in the event of a real-world side collision any modification and extra design for survival chance in a real-world collision and/or on rollover will generate costs, R&D (Research and Development) expenses and weight due to the use of other inventions.

Summary of the advantages of the present invention:

A) saving labour-time by installing and adjusting engaging parts from outside the vehicle body.

B) low reject rate.

C) space-saving, inexpensive design.

D) dissimilar operating planes or at least two operating planes for each vehicular couple to ensure the engagement of its interengaging assemblies in association with energy absorption due to load cases in three different planes. Figs. 14 to 18 illustrate *a single vehicular couple*: window-guide channel & B-pillar with the interengaging assemblies: keys 34 & holes in z-x plane acting as the first operating plane, however, interengaging assemblies: keys 32, 33 & holes in z-y plane acting as the second operating plane. The specification is changed from the minimum tolerances of "narrow" to permissible tolerances of "far less narrow", thus cutting costs and time associated with less adjustment work to reduce large clearances thereto. This feature of dissimilar operating planes is applicable too for both interengaging assemblies: holes & 15.1, 15.2a and 15.2, 15.3 and 15.4a, 15.5 etc., shown in Fig. 3. A row of the same keys is operative in dissimilar operating planes by arranging a number of the same keys 15.1 to the generally inclined A-pillar or of keys 33 to the generally inclined B-pillar. In reference to the global xyz coordinate system the key 15.2a & hole is operative in an inclined plane. Because the hinge bolts of the front and rear doors have an operating direction in z-axis the arrangement of interengaging assemblies: holes & keys 31, 36 to one operating plane is sufficient. However, any additional arrangement of holes & keys 30, 35 improves the engagement of vehicle mating parts and substantially decreases severe/fatal injuries in any real-world collision.

E) minimizing the R&D work by reducing FEM calculations, crash tests and by saving material due to the arrangement of interengaging assembly in different operating planes.

F) passenger protection for all collisions by a **single** construction, manufacturing, testing expenditure, assembly and material supply.

G) exploitation of the flange 21, 21T, 21h, 21x of vehicle body 20 provided with sound-proofing material 21.10, shown in Figs. 1, 17, 18, due to the sites to accommodate keys and the continuous stress curve. The enlargement of the flange to a limited extent neither impairs the overall stylish impression nor obstructs the passenger from ingressing into or egressing from the passenger compartment. Those regions of all pillars are defined by the dotted lines "a1", "b1", "b2" and "c1".

H) overall stylish impression. As substitutes of the bulky bolt ref. to U.S. Pat. No 3,819,228 small-size parts can be distributed in inconspicuous manner along the window-guide channels as well as flange, thus substantially ensuring the engagement of vehicular couple whilst lowering stress. Due to this feature it is possible to arrange the following keys:

- 30, 32, 35, 37 to the respective flange 21 of vehicle body 20. In contrary to U.S. Pat. No. 3,819,228, this feature won't endanger passenger when stepping in or out, furthermore, more useful for passenger protection in a side collision, particularly,



according to collision types „U1” and „U2”, shown in Fig. 13, as well as in a front collision.

- 15.2a, 15.2, 15.7 e.g. with bolts M4 to the narrow window-guide channel 6.3, 6.3B of upper member 8.17 of door frame to resolve the problem of the large, stiff contour groove of the prior art.
- 33, 34, 36 to the respective window-guide channels 6, 6B and channels 6.7, 6.8 in engagement with the reinforced B-pillar in two to three operating planes without obstructing the operation of the seat belt 26.1, shown in Fig. 15. The fact, that no contact is made during the opening operation of series-connected vehicle doors, is demonstrated by the trajectories of both outer points of the washer and of the door edges drawn with dotted lines.
- 31 to the respective window-guide channels 6 and channels 6.6a in engagement with the reinforced A-pillar.

I) less stress to solve the problem of total deformation. By means of arrangement of interengaging assemblies of each vehicular couple in multi-operating planes and increase of vehicular couples comprising vehicle door & vehicle roof 17, vehicle door & side rail 18, vehicle door & pillar/s and vehicle door & vehicle body 20 more vehicular members in compound construction are involved in energy absorption in different load cases in the event of any collision and/or rollover.

In co-operation with another prior art the structural stiffness reaches the maximum.

Beyond doubt, the advantage of keys 2.1, 5.6 & mating holes is due to the further exploitation of the very stiff impact beams 1, 7 to house the corresponding parts.

Because the other vehicular couples comprising such as vehicle door & side rail and vehicle door & vehicle roof are not equipped with interengaging assemblies this *single* arrangement of one vehicular couple in mid-region of door is insufficient in the event of any collision and/or rollover, therefore endangering the passengers in the following state of deformation

- intrusion of vehicle roof 17 into the vehicle body and of upper member 8.17 of door frame, thus squashing the passengers and
- buckling of the upper portion of the A-pillar, total deformation of upper member 8.17 of door frame, buckling of vehicle roof 17 and buckling of side rails 18, shown in Fig. 8.

In order to avoid the above-mentioned state a number of holes or keys 30 to 37 is arranged to the flange 21 *above, below* of the impact beams 1, 7 and *therebetween*.

When the *non-adjustable* rivets 5.6 of the door hinges in x-z operating plane are replaced by a number of interengaging assemblies 15.1, 15.2a, 15.4, 30, 31 in numerous operating planes, the total stress of the vehicular couples: A-pillar & vehicle door along the z-axis is lower owing to stress distribution, thereby preventing, to a certain extent, the A-pillar and vehicle door from total deformation and gap „o”, shown in Fig. 8.

J) measures against passenger ejection and total deformation of the vehicular members, whereby vehicle doors are not or less deformed, in real-world accidents ref. to problem cases E2 to E4, which can solely be solved by engagement of the following interengaging assemblies governed by permissible tolerances:

- holes & keys 15.3, 15.3a, 15.5a, 15.5 owing to U-shaped extension members 17.3, 18.3, whose deformation causes a constrained deformation of the series-connected vehicle doors, vehicle roof and side rails;
- holes & keys 32, 33, 34, 30, 15.2, 15.4a of the vehicular couple comprising vehicle door & B-pillar in four operating planes; *and/or*
- hooks 15.6 & reinforcing rod 17.1d of both vehicular couples comprising series-connected vehicle doors & side rail and series-connected vehicle doors & vehicle roof,

so that the deformation of the side rail and vehicle roof causes a constrained deformation of the series-connected vehicle doors; and by *energy transmission* into the other vehicle side by means of transverse girders 17.2, 17.2b, 17.2c, 17.2d, 18.2 of vehicle roof, side rails and all pillars facing each other, thus distributing the energy thereto.

K) passenger protection by engagement of vehicle couples in rear collision. Door detachment in rear collision occurred due to the lack of door hinges and interengaging assemblies. For the purpose of connection of vehicular members to each other the engagement of rear door 8B with the C-pillar is improved by rigidly arranging

- member 6.5C, adapted to the outer door-contour and having holes to receive mating keys 37, shown in Figs. 14, 18, to the door frame of rear door; and
- keys 33, 34 to window-guide channel 6B.

The features of vehicle door are, doubtless, suitable for tailgate door 8T, sliding side door, liftgate door cargo door, trunk cover 8x, hood 8h, series-connected doors, e.g. three vehicle doors with four pillars of large van.

### BRIEF DESCRIPTION OF THE DRAWINGS

A number of embodiments, other advantages and features of the present invention will be described in the accompanying drawings with reference to the xyz global coordinate system:

Fig. 1 is a side view of vehicle side, body, impact beams, keys, hooks, window-guide channels serving as reinforcing elements.

Fig. 1A is a cross-sectional view of a vehicle door engaging with a roof and side rail ref. to DE-OS 2162071 in side collision.

Fig. 1B is a cross-sectional view of a vehicle door engaging with a side rail ref. to EP 0423465 A1 in side collision.

Fig. 2 is a side view of an U-shaped window-guide channel, the position of keys 15.7, 15.8 and of an additional window-guide member 6.4, 6.4B.

Fig. 2A is a side view of an U-shaped window-guide channel, the position of keys 15.7.

Fig. 3 is a perspective view of a front stiff door frame with both window-guide channels, both respective window-guide channels and interengaging assemblies of the 1st embodiment.

Fig. 3A is a cross-sectional view of a key equipped with an adjusting mechanism.

Fig. 4 is a perspective view of interengaging assembly hooks & reinforcing rod of the 2nd embodiment.

Fig. 4A is a cross-sectional view of the reinforcing rod and the mating hook equipped with an adjusting mechanism.

Fig. 5 illustrates a load case I in z-y plane in front collision of vehicle.

Fig. 6 illustrates a load case II in z-x plane in front collision.

Fig. 7 illustrates a load case III in x-y plane in front collision.

Fig. 8 is a state of total deformation of vehicle at displacement „v” in front collision.

Fig. 9 illustrates a load case IV in x-y plane in side collision of vehicle.

Fig. 10 illustrates a load case V in z-x plane in side collision.

Fig. 10A illustrates the mating parts of interengaging assemblies ref. to U.S. Pat. No 4,307,911, both mating parts of a door lock, the general force „F<sub>1</sub>” or „S<sub>1</sub>” in the event of front or side collision and a highway column.

Fig. 11 is a view of a compression-coil spring on a lower spring seat.

Fig. 12 illustrates the projection of the end coil and spring seat in a plane, the test results and FEM data of an end coil rolling on the lower spring seat in dependence on load.

**Fig. 13** illustrates four collision types „U1” to „U4” ref. to the research work of Institute of Vehicle Safety, a Dept. of German Insurers Association, and a highway column.

**Fig. 14** is a perspective view of interengaging assemblies of the 3rd embodiment comprising a stiff front door frame having a single window-guide channel and a stiff rear door frame having a single window-guide channel to engage with the pillars and flange of vehicle body.

**Fig. 15** is a cross-sectional view of the series-connected doors in engagement with the A-, B-pillar and of the vehicle body along the line D-D in **Fig. 14**.

**Fig. 16** is a side view of the series-connected stiff door frames without window pane in engagement with the B-pillar according to arrow E in **Fig. 14**.

**Fig. 17** is a perspective view of interengaging assemblies of the 4th embodiment comprising a stiff front door frame having a single window-guide channel in engagement with the flange of vehicle body.

**Fig. 18** is a side view of the flange of vehicle body provided with keys.

### DESCRIPTION OF THE PREFERRED EMBODIMENTS

Beyond doubt, the function of the interengaging assemblies is well described in the preferred embodiments of the prior art. However, the tolerances, e.g. eight tolerance zones of each interengaging assembly ref. to U.S. Pat. No. 5,806,917, are totally neglected in the scope because the explanation of how to assemble and manufacture the interengaging assemblies in relation to the **Figs.** is omitted. Hence, this subject must be taken into account when the function and assembly of the interengaging assemblies is described in conjunction with manufacturing parts thereof, distributing energy to the vehicular members and increasing the vehicle stiffness. When the door is closed, the interengaging assemblies in engagement is capable of ensuring the connection of the door with the vehicle body when the clearances (tolerances) between the mating parts thereof are well defined.

Ref. to **Fig. 3**, the scope of the application of the window-guide channels of vehicle door is extended to accommodate the keys of interengaging assemblies, whose mating receptacles are arranged to any (A-, B-, C- or D-) pillar, flange of vehicle body, vehicle roof and/or side rail. This feature saves weight and costs. The positions of keys and mating receptacles may be interchanged if desired.

According to the prior art a stiff door frame of vehicle door can be assembled from at least two impact beams provided with interengaging assemblies reinforcing elements and at least one conventional window-guide channel, shown in **Figs. 1** and **3**, which is made from U-shaped thin panel with low-grade tensile. Contrarily, the window-guide channels **6**, **6B**, **6.1**, **6.2**, **6.1B**, **6.2B**, **6.1a**, **6.2a**, **6.1aB**, **6.2aB** are of high-grade tensile strength to receive keys (hooks), mating receptacles, retaining members **6.5**, **6.5B**, **6.6a**, **6.6b**, **6.7a**, **6.7b**, **6.8**, **6.9** (not drawn), reinforce door frame, span the door aperture and distribute impact energy to the vehicle roof, side rail and pillar(s).

The retaining members **6.8**, **6.9**, shown in **Fig. 14**, are fixedly attached to the front faces of both impact beams **1B**, **7B** and window-guide channel **6B**, the retaining members **6.6b**, **6.7b** to window-guide channel **6** and impact beam **7** and the retaining members **6.6a**, **6.7a** to window-guide channel **6** and between both impact beams **1**, **7**.

Both window-guide channels are replaceable by a conventional U-shaped stiff window-guide channel **6**, **6B**, shown in **Figs. 2**, **2A**, **14** to **17**. Less stiff window-guide members **6.3**, **6.3B** are normally made of panel. Alternately, very stiff window-guide member **6.3**, **6.3B** serves to receive the window pane and keys **15.7**.

Window-guide channel 6, 6B provided with window-guide member 6.3, 6.3B in the door cavity, shown in Fig. 2A. has open ends. To maximize the stiffness of window-guide channel 6, 6B both ends are rigidly connected to each other by window-guide member 6.4, 6.4B in the door cavity, shown in Figs. 2, 14 to 17, after the window pane has been  
5 inserted. Alternately, the window-guide member 6.4, 6.4B, having flat profile, receives window pane 60, 60B, shown in Fig. 15, and, finally, is secured against falling down by protective parts.

The window-guide member 6.4, 6.4B is useful for the accommodation of keys 15.8. If extraneous weight is not that important for heavy cars, trucks and vans, the window-guide  
10 channel, fastened to the impact beams, serves as parts of door frame to receive keys while guiding and receiving the window pane.

One of the solutions for the problem case E4 and energy-distribution to pillar, door 8, 8B, roof 17 and side rail 18 as well as from one vehicle side to the other vehicle side is featured in the 1st embodiment, shown in Fig. 3, by arranging keys 15.1 to reinforced A-pillar and  
15 the mating oblong holes to window-guide channel 6.1a, keys 15.2 to window-guide channels 6.1a, 6.2a and the mating holes to reinforcing panel 17.1a arranged along the vehicle roof, keys 15.4 to the reinforcing plate of reinforcing panel 18.1 arranged along the side rail, and the mating holes to window-guide channels 6.1a, 6.2a and key 15.1 to a  
20 reinforcing element of the L-shaped A-pillar, welded to reinforcing panel 17.1c arranged along the vehicle roof and to transverse girder 17.2d of both facing A-pillars of both vehicle sides, and the mating oblong hole to window-guide channel 6.1a.

The load case IV becomes apparent when large-sized doors are designed to enhance the comfort of passengers when stepping in and out of the vehicle. To resolve this drawback additional keys 15.2, 15.4 are arranged to window-guide channel 6.3 and member 6.4 and  
25 the mating holes to the reinforced vehicle roof and the reinforced side rail, respectively.

Ref. to Fig. 4, the 2nd embodiment consists of an interengaging assembly, the hooks of which are attached to two window-guide channels of each vehicle door and the mating rod to the vehicle roof, pillars of the door or all doors. Additionally, the rod serves to reinforce  
30 the vehicle roof, sustain impact force and aid positioning on assembly, thus cutting costs and time at the assembly line. However, this embodiment needs space, which is available in large cars, trucks and vans. This embodiment is suited too for another vehicular couple comprising vehicle door/s & side rail.

The interengaging hooks 15.6 are bolted to window-guide channels 6.1a, 6.2a, 6.1aB, 6.2aB and the mating reinforcing rod 17.1d is arranged along the vehicle roof 17 and/or  
35 side rail 18. When at least one pair of rods is welded to transverse girders 17.2e, 17.2f, 17.2g of both A-, B- and C-pillars, energy can be distributed from one vehicle side to the other vehicle side in a side collision, from the front to rear vehicle section of vehicle body 20 in a front collision, from the rear to front vehicle section of vehicle body 20 in rear collision or to all members of vehicle body 20 on rollover.

Ref. to Figs. 14, 17, 18. the 3rd embodiment consists of interengaging assemblies 30 & 6.5, 35 & 6.5B and other interengaging assemblies 32 & 6.9, 37 & 6.9B (6.9, 6.9B similar to 6.5), 37 & 6.5C for the purpose of avoiding large deformation of the edges of each door and of saving costs by exploiting the flange 21 of vehicle body 20 and the enlarged flange defined by the dotted lines "a1", "b1", "b2" and "c1" to receive keys. The keys 30, 32, 35,  
40 37 are bolted to the respective reinforcing elements 21.1 to 21.5, 21.1B to 21.5B of the flange 21 of vehicle body 20 and the corresponding holes are arranged to the housings 6.5, 6.5B and/or auxiliary member 6.5C, all of which are rigidly attached to the respective window-guide channels 6, 6B, the respective members 6.6b, 6.7b, 6.8, 6.9 (not drawn

because of the similarity to 6.7b) and/or the respective impact beams 1, 1B, 7, 7B. The reinforcing element 21.5B is welded to the flange and rear wheel case. The same reinforcing method can be employed to arrange a similar element 21.1 to the flange and the front wheel case.

5 Stiff door hinges in co-operation with impact beams 1, 7, 1B, 7B and interengaging assemblies transmit forces of load case I from the front to rear vehicle section of vehicle body 20 in a front collision. There are no door hinges to connect the rear door to the C-pillar. To avoid the detachment of rear doors and improve energy transmission from the rear to front vehicle section of vehicle body 20 in rear collision, an auxiliary member 6.5C is  
10 attached to the impact beams 1B, 7B.

Instead of the bulky "engaging" bolt ref. to U.S. Pat. No. 3,819,228 these keys, configured in small size and distributed along the flange, neither spoil the overall design nor injure persons, stepping in or out of the vehicle body, nor overstressing the vehicle members.

15 In the 4th embodiment the Technical Mechanics Method of constrained deformation is applied to ensure the engagement of all vehicular members with each other in the event of accidents and to distribute impact energy thereto by means of two U-shaped extension members 17.3, 18.3, located in common pillar, shown in Fig. 3, whose keys 15.3, 15.3a, 15.5, 15.5a are engaged with the mating apertures, arranged to the corresponding window-guide channels 6.2a, 6.1aB of series-connected doors 8, 8B, when doors are closed.

20 Serving as a connection element of the common pillar with the vehicle roof and series-connected doors, this U-shaped extension member 17.3 is welded to reinforcing panel 17.1b, arranged along vehicle roof 17, and to transverse girder 17.2c of both facing common pillars of the vehicle sides. Serving as a connection element of the common pillar with the vehicle floor and series-connected doors, this U-shaped extension member 18.3 is  
25 welded to reinforcing panel 18.1b, arranged along the vehicle floor, and to transverse girder 18.2 of both facing common pillars of the vehicle sides. The belt case 26 can be housed in the U-shaped extension member 18.3.

When in the real-world accidents, above-mentioned in the problem cases E3 and E5, the side rail 18 or vehicle roof 17 deflects inwards, conventional interengaging assemblies  
30 become disengaged. Large inward deflection of the side rail or vehicle roof deforms the extension member 18.3 or 17.3, which constrained deforms the common pillar, series-connected doors and the vehicle floor or the vehicle roof thus ensuring the engagement of the series-connected doors with the vehicle body and preventing passengers from being hurled out of the vehicle, when rolling over.

35 Due to the arc-travel path of the door about the mutual axis of door hinges the mating surfaces of key and receptacle of each interengaging assembly, proposed by U.S. Pat. No. 5,806,917, are configured in four tapered forms or two curved and two tapered forms, thus yielding eight tolerance zones, high manufacturing and assembling costs as well as making tight engagement impossible resulting in door detachment in accidents. To resolve these  
40 problems straight (non-curved, non-inclined or non-tapered) engaging surfaces are proposed for key and receptacle. The purpose of assembling and adjusting any key, shown in Figs. 3, 3A, 4 and 4A, from outside of the vehicle body 20 is to substantially cut labour time and costs. Manufacturing costs can be enormously lowered by using mechanical standard parts like washer, hexagon socket head bolt etc. With the exception of 15.4a each  
45 key 15.1 to 15.5a, 15.7, 15.8, 30 to 37 comprises a bolt 15.14, a sleeve 15.11, a number of washers built into one spacer 15.12 and a washer with a large exterior diameter 15.13, illustrated in Figs. 3A, 14 to 18.

In order to ensure the engagement of key with mating hole a protrusion „ $x_n$ ” and circumferential clearance „ $c_c$ ”, explained in the next section, must be preserved by correcting the length of spacer „ $l$ ” by removing or adding washers and/or assembling a sleeve with exterior diameter „ $d$ ”, washer with exterior diameter „ $D$ ” and/or spacer with diameter „ $d_R$ ”.

If desired, the sleeve 15.11 and spacer 15.12 can be made of soundproofing material.

Each hook 15.6, shown in Figs. 4 and 4A, comprises a hook 15.20 with interior diameter „ $d_1$ ” and gap „ $s_1$ ”, smaller than „ $d_1$ ”, a bolt 15.21, a number of washers built into one spacer 15.22, a coil-spring washer 15.24 and a nut 15.25. The symbols „ $s_1$ ”, „ $d_1$ ” and „ $d_2$ ” are shown in Fig. 4A. In order to ensure perfect engagement of the hooks with reinforcing rod 17.1d, having diameter „ $d_2$ ” smaller than „ $s_1$ ”, small tolerance zones, shown in Fig. 4A, must be preserved by assembling a hook with gap „ $s_1$ ”, assembling a rod with diameter „ $d_2$ ”, correcting the distance „ $l$ ” by removing or adding washers and/or positioning the centres of the hook hole and the reinforcing rod out of alignment.

Advantageously, the rod 17.1d, reinforcing the side rail 18 or vehicle roof 17, enhances the stiffness of vehicle structure and avoids in large extent buckling thereof.

Fig. 15 exemplifies another feature of numerous different planes, wherein the interengaging assemblies of any vehicular couple comprising e.g. the common or B-pillar and the series-connected vehicle doors 8, 8B, operate. When the doors are closed, key 33 protrudes the mating hole by „ $-x_n$ ” (minus sign in respect to the opposite x-direction), which is limited due to the arc-travel path of the door about the axis of door hinges. The clearances of key 33 and the mating hole are denoted by „ $-y_n$ ” and „ $y_p$ ”. The protrusion „ $x_n$ ”, circumferential clearance „ $c_c$ ” (not drawn, represented by „ $-y_n$ ” and „ $y_p$ ” in y-direction) of the mating parts of each assembly and operating plane play a significant role on tight engagement thereof in accidents. In the real-world accidents, above-mentioned in the problem cases E2 and E3, the door becomes detached due to large circumferential clearances of all mating parts of interengaging assemblies, which operate in the same z-y plane, and large inward deflection of the vehicle body 20 or side rail 18 in the opposite x-direction, during which under the load of inertia forces of the passenger the door is opened and moved in the arc-travel path about the axis of door hinges. Door detachment can be prevented by minimum tolerances, whereby the mating parts of interengaging assemblies of any vehicular couple, acting in the same operating plane, are governed.

In this time- and cost-saving feature against door detachment, proposed for the following embodiments, many interengaging assemblies of any vehicular couple comprising e.g. interengaging assemblies keys 32, 33, 34 & mating holes, must operate in numerous different planes, where the deformation of door 8 results in a tight engagement of keys 32, 34 with the mating holes. taken, the worst case is given, that all keys 33 fail to engage with the mating holes. The interengaging assemblies, comprising keys 32, 33, 34 & mating holes, operate in three different planes, the number of which can be increased by arranging these interengaging assemblies in the planes, which, however, are offset to each other, e.g. in offset z-y planes. The interengaging assemblies keys 35 & holes act in the fourth operating z-y plane and keys 36 & holes in the fifth operating z-x plane. Owing to this feature the minimum tolerances of "narrow" are outdated, hence, replaced by permissible tolerances of "less narrow", "far less narrow", "small" and/or "medium", thus significantly lowering the reject rate, assembly time and costs. Advantageously, a pattern of the interengaging assemblies, governed by permissible tolerances, can be issued in a table handed to assembly workers. Alternately, this pattern can be coded in the assembly program to drill, position and assemble parts thereof within the permissible tolerances.

The constant, small contour clearance and the proper tolerance between door lock 248 and striker 298, above-mentioned in the problem cases D and E, can easily be accomplished at the assembly line within short time, thus making rework as well as adjustment work superfluous.

5 It must always be reckoned with a reject when the assembly tolerances are, unexpectedly, larger than the permissible tolerances. Adjustment work for the interengaging assemblies of the rejected car can be done outside of the assembly line, thereby the production process is not halted and the reject rate of zero is met.

10 All these advantages outweigh the costs of extra material for a larger number of interengaging assemblies.

A washer 15.13 with radial teeth, serving as part of key 33, clamps in the inner region of the reinforced B-pillar in any collision or on rollover. As an integral part of a bolt the washer won't come loose on assembly.

15 Costs can be cut by positioning an unadjusted key between adjustable keys, such as rivet 15.4a, fastened to the reinforcing plate of reinforcing panel 18.1a arranged along the side rail. However, when the number of the interengaging assemblies is limited in a low-cost configuration, for perfect interengagement the provision with keys 15.1 to 15.8, 30 to 37 without key 15.4a is ultimately necessary.

20 Large total stress of the load cases e.g. I to III results in total deformation (buckling) of the pillars, side rail, vehicle roof and/or doors because stress of vehicle body and doors in a real-world accident can never be predetermined in the research and crash tests, three of which are mentioned in the problem case E4, due to the collision type, the boundary conditions and properties of two masses colliding against each other. Four front collision types are shown in Fig. 13. In a real-world accident a front, side and/or rear collision can  
25 end up in a pile-up or on a rollover, thus increasing the number of collision types and making a FEM calculation impossible. To resolve such indeterminate stress the vehicular couples comprising front pillar & door 8, 8B, rear pillar & door 8, 8B, vehicle roof 17 & door 8, 8B and side rail 18 & door 8, 8B must be equipped with many interengaging assemblies operating in numerous planes, such as keys 30 & holes acting in the first  
30 operating z-y plane, keys 31 & holes acting in the second operating z-x plane, key 15.2a & hole, shown in Fig. 3, acting in the third operating z-y plane and in co-operation with additional interengaging assemblies, the mating parts of which may be chosen among the keys 15.1, 15.2, 15.3, 15.3a, 15.4, 15.4a, 15.5, 15.5a, 15.6 to 15.8, 32 to 37 and mating receptacles in the above-mentioned embodiments.

35 Although the present invention has been described and illustrated in detail, it is clearly understood that the terminology used is intended to describe rather than limit. Many more objects, embodiments, features and variations of the present invention are possible in light of the above-mentioned teachings. Therefore, within the spirit and scope of the appended claims, the present invention may be practised otherwise than as specifically described and  
40 illustrated.